

# Elements of entrainment

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## Abstract

A unified model is proposed for the fundamental behavior of turbulent entrainment over a broad class of flows. The entrainment velocity is expressed as the ratio of a relevant length scale to a time scale for all flows, a generalization of the original entrainment hypothesis of Morton, Taylor, & Turner. This generalization appears to bring the theoretical effects of acceleration, compressibility, confinement, rotation, stationarity, and stratification in accord with observation.

## Introduction

Turbulence has been called the most important unsolved problem in all of classical physics. From astrophysics to oceanography, aeronautics to combustion, turbulence is ubiquitous. Yet in spite of its central role in science and engineering, turbulence has defied solution for over a century.

The most important property of turbulence is entrainment. Both transport and mixing in turbulent flows are controlled by entrainment. Boundary layer heat transfer and skin friction are the transport of energy and momentum at a wall. The vertical transport of water and energy in the atmosphere and ocean are determined by stratified entrainment. In high Reynolds number flow, the mixing is entrainment-limited, so much so that the molecular diffusivity can change by three orders of magnitude while the molecular mixing rate changes by only a factor of two. Entrainment determines most of what we really want to know about a turbulent flow.

## Entrainment hypothesis

Half a century ago, Morton, Taylor, & Turner (1956) proposed the most successful hypothesis for entrainment. In order to model a thermal rising from the sudden release of buoyant fluid, they argued on dimensional grounds that the local entrainment velocity  $v_e$  into the thermal at any station must be proportional to the rise speed  $W$  of the thermal at that station. There is simply no other speed available on which to base the entrainment velocity (see figure 1). In this way, the thermal grows linearly with height, in accord with observation. Furthermore, their hypothesis is equally valid for a wide variety of other classical flows that might be termed “ordinary turbulence,” correctly accounting for the entrainment rate in the plume, shear layer, jet, wake, etc.

However, the entrainment hypothesis sometimes fails. For example, when the speed of sound becomes comparable to the velocity jump across a shear layer, the entrainment rate

precipitously declines by a factor of five (Papamoschou & Roshko 1989). This cannot be explained by the original entrainment hypothesis. The entrainment rate is also strongly affected when acceleration, confinement, rotation, or stratification become appreciable. This paper is an attempt to extend the entrainment hypothesis into a more general theory.

### Entrainment process

Entrainment was thought to be a small-scale nibbling process at the edge of a turbulent region. Corrsin & Kistler (1955) proposed a “superlayer” there, across which fluid was thought to be entrained by small-scale nibbling. Shadowgraph images of the supersonic round wake of a projectile seemed to support this notion. However, shadowgraph images of the plane shear layer revealed the engulfment of large tongues of fluid by the largest vortices in the flow (Brown & Roshko 1974, Roshko 1976). The two-dimensional geometry of their shear layer allowed a more clear view of the entrainment process. Instead of polite little nibbles, their images revealed that the turbulence really entrains like a hungry teenager taking big gulps of fluid. These large engulfed tongues of pure, unmixed fluid are transported by the large-scale vortices entirely across the layer (Konrad 1976).

### Entrainment rate

The entrainment rate  $v_e$  is a velocity. From dimensional considerations, it must therefore always be expressible as the ratio of a relevant length scale to the rotational period  $\tau_\lambda$  of the eddy responsible for entrainment. If there is engulfment, then the relevant length scale must be the size  $\lambda$  of the entraining eddy.

$$v_e = \text{const.} \frac{\lambda}{\tau_\lambda}. \quad (1)$$

Of course, the dimensional argument cannot establish the value of the constant of proportionality. If there is no engulfment, such as at a solid wall or at a strongly stratified interface, the length scale must be a diffusive one, the square root of the product of the diffusivity and an eddy time.

For ordinary, incompressible, free shear flows, the entrainment rate must be proportional to the ratio of the size of the largest eddies to their rotation period. This is a direct consequence of Roshko’s engulfment, whereby the first step of engulfment by the largest eddies is rate-limiting. The subsequent processing of the engulfed fluid by all smaller eddies is both proportional to and sufficiently fast compared to the largest eddies that only the largest eddies matter. Since the largest eddies control the rate, we do not need to know much about anything else. This happy circumstance vastly simplifies matters, such as modeling the mixing (Broadwell & Breidenthal 1982).

So for such flows equation (1) becomes

$$v_e = \text{const.} \frac{\delta}{\tau_\delta}, \quad (2)$$

where the subscript  $\delta$  is the size of the largest eddies. Since the characteristic velocity of the turbulent flow is also proportional to  $\frac{\delta}{\tau_\delta}$ , we recover the Morton *et al.* entrainment hypothesis for ordinary turbulence.

As indicated above, equation (2) does not always work. Let us now consider the various violations of the entrainment hypothesis.

### Acceleration

Like people, ordinary vortices slow down as they age. That means that the rotation period of the largest eddies  $\tau_\delta$  increases with time  $t$ . In self-similar turbulence, there is no other distinguished time scale, so the period must be proportional to the age of the vortex from its virtual origin.

$$\tau_\delta(t) = \text{const.} t \quad (3)$$

For all ordinary turbulence, the constant of proportionality is positive, as is confirmed by examination of the observed growth laws of these flows. Their rotation period always increases with age. These flows are termed “unforced”.

Note that the vortex rotation period in an unforced flow may not exactly follow equation (3) over a short time interval. For example, the large-scale vortices in the free shear layer obey equation (3) in the long term, but on time scales less than the pairing time, a vortex does not necessarily follow (3). We will ignore this subtlety here.

### *forced turbulence*

However, it is possible to force the flow in such a way that the rotation period does not increase with age. Define an acceleration parameter  $\alpha$  such that

$$\tau_\delta = \tau_0 - \alpha t, \quad (4)$$

where  $\tau_0$  is the large-eddy rotation period at the arbitrary time  $t = 0$ . For ordinary, unforced turbulence,  $\alpha < 0$ . If the flow is forced,  $\alpha$  can be zero or even positive.

### *temporal self-similarity*

The vortices are temporarily self-similar if their next rotation period is proportional to their last one. Otherwise there would be a special, distinguished time scale, a contradiction of self-similarity. Figure 2 illustrates the evolution of the rotation period of temporally self-similar turbulence. The line must be straight and  $\alpha$  must be a constant. For all ordinary, unforced turbulence,  $\alpha < 0$  and slopes upward.

### *exponential jet*

The line is horizontal if the next rotation period is the same as the last ( $\alpha = 0$ ). This can be achieved in an exponential jet, where fluid is ejected from a nozzle with a speed  $V_J(t)$  that increases exponentially in time,

$$V_J(t) = V_{J0} e^{t/\tau_e}, \quad (5)$$

where  $V_{J0}$  is the nozzle speed at  $t = 0$ . Because of this forcing, every large-scale vortex in the exponential jet rotates with the same period, equal to the e-folding time  $\tau_e$  imposed on the flow, no matter how old or how far from the nozzle. The vortices never age. It is a kind of perpetual youth.

Remarkably, acceleration reduces the normalized entrainment rate. A convenient way to measure entrainment at large Reynolds number is with a fast chemical reaction that destroys a visible chemical in the nozzle fluid when mixed with the ambient fluid. If the mixing is entrainment-limited, changes in the visible “flame length” reflect changes in the normalized entrainment rate. Compared to the ordinary jet, the exponential jet has about a 20% greater flame length (Kato *et al.* 1987). In fact, such acceleration is the only known method for affecting the far-field entrainment rate of the incompressible jet, as noted by Zhang & Johari (1996). Their detailed images of jets with modulated nozzle speed demonstrate that acceleration only influences the entrainment rate when the imposed change in velocity during one vortex rotation is comparable to the initial velocity. In other words, the logarithmic derivative must be appreciable.

### *super-exponential forcing*

The third category is the line sloping downward in figure 2 ( $\alpha > 0$ ). In spite of getting older, the vortices spin ever faster. After a finite time, the spin rate becomes infinite and the rotation period vanishes.

One might anticipate that the entrainment rate would be further reduced as  $\alpha$  increases. Using dimensional and heuristic arguments, one theory has been proposed (Breidenthal 2003 with different notation). The dimensions of the dissipation rate per unit mass are  $(\text{length})^2(\text{time})^{-3}$ . Every canonical turbulent flow has a conserved quantity  $Q$ . For example, in the shear layer, it is the velocity difference  $\Delta U$ . If the dimensions of  $Q$  are in general

(length)<sup>m</sup>(time)<sup>-n</sup>, the dissipation rate is proportional to  $Q^m \tau_v^{-\left(3-\frac{2n}{m}\right)}$ , where the vortex period is  $\tau_v$ .

For super-exponential forcing,

$$Q = Q_0 e^{\frac{t}{\tau_0 - \alpha t}}, \quad (6)$$

where  $Q_0$  is the value of  $Q$  at  $t = 0$ .

Define  $D$  to be the dissipation rate normalized by that of the unforced flow. From heuristic grounds, we conjecture that the quantity is the natural scaling of effect of  $\alpha$  on  $D$ . If so, then

$$\beta \equiv -\left(3 - \frac{2n}{m}\right) \frac{dD}{D} = -\frac{d\alpha}{\left(3 - \frac{2n}{m}\right)} \quad (7)$$

$$D = e^{-\frac{\alpha - \alpha^*}{\beta}}, \quad (8)$$

where  $\alpha^*$  is the value of  $\alpha$  for the unforced flow.

## Compressibility

It has long been known that a compressible flow grows more slowly than an incompressible one. Papamoschou & Roshko (1989) found that the spreading angle of a turbulent shear layer dropped by a factor of about five as the Mach number increased. Linear stability theory may provide an indication of the entrainment behavior, since the underlying instabilities drive the basic flow. However, the indication can only be qualitative, in as much as the finite amplitude eddies are fully nonlinear.

Bogdanoff (1983) recognized that the important parameter for the instability is a “convective” Mach number, the Mach number of the outer flow with respect to the speed of the instability waves. A hint that this is the correct approach comes from the flow models of Brown (1974), Coles (1981), and Dimotakis (1986), discussed below.

One heuristic model that addresses the fully nonlinear flow supposes that nonsteadiness is essential to entrainment. This is a hint of this in the results of the Oster-Wygnanski (1982) experiment, where the vortices in a shear layer are forced to be equally spaced. For a certain time, these vortices are steady, resembling Kelvin’s cat’s eye pattern (Kelvin 1880), with no

vortex pairing. Remarkably, Oster & Wygnanski found that the Reynolds stresses vanish. There is no turbulent transport of momentum. Roberts (1985) found the mixing rate essentially vanishes, in spite of the fact that the vortices are continuing to rotate. If nonsteadiness is required for entrainment, it follows that the signaling speed of acoustic waves must control the physics, since the information about a nonsteady event can travel no faster than the speed of sound.

There is a subtle point to note here. Mach number plays two simultaneous and different roles in high speed flow (Roshko, private communication). On one hand, it indicates the signaling process above. It is also a measure of the energy content of the flow, i.e. thermal vs. kinetic. Indeed, most attempts to model compressibility have focused on energy and density considerations.

The second assumption is that the important time scale for an eddy to entrain is always about one vortex rotation. This is the behavior of the engulfment and mixing process in incompressible turbulence (Brown & Roshko 1974). The immediate consequence of these two assumptions is that entrainment is controlled by a “sonic eddy” whose rotational Mach number is unity (Breidenthal 1992). Such an eddy completes one rotation during the signaling time across its diameter. Any larger eddies that might exist would play no role in the entrainment process whatsoever.

The hypersonic wake provides a good opportunity for comparison with experiment. The model predicts that the initial wake growth rate should be zero, since the large-eddy rotational Mach number is greater than unity there. Only sonic eddies, much smaller than the total wake thickness, are capable of transporting momentum. The time scale for the sonic eddies to transport momentum across the entire wake is the square of the wake thickness divided by the product of the speed of sound and the sonic eddy size, this product being the effective turbulent diffusivity. Note that the concept of turbulent diffusivity is rarely justified.

The initial wake should not grow at all until the rotational Mach number of the largest eddies has fallen to unity. Then the growth rate should transition to the incompressible value. The time for this transition is set by the transport of momentum by the sonic eddies across the width of the wake. Since they are small compared to the width of the wake, the process can be modeled by turbulent diffusion, with a diffusivity equal to the product of the speed of sound and the size of the sonic eddy. Note that for most flows, turbulent diffusion is not an appropriate model (Corrsin 1974). Only in the rare circumstance of the entraining eddies being small compared with the distance in question is diffusion a reasonable model.

The transition is predicted to occur at a downstream station of  $M^2d$ , where  $d$  is the effective body diameter. At  $M = 20$ , this would be 400 effective body diameters downstream, which is in accord with shadowgraph observations (Finson 1973).

## Confinement and mixing

When engineers mix chemicals together, they usually want to retain the mixture in a confined chamber. Examples include combustion and chemical processing. So we will generalize the

term entrainment here to include the entire physics of transport and molecular mixing in a confined vessel.

Consider the probability density function (pdf) for the concentration of an inert scalar mixing with a second fluid in some general flow sketched in figure 3. Initially the pdf consists of two delta functions at the extrema, corresponding to the two pure fluids. As the turbulence mixes some of the two pure fluids together at intermediate concentrations, forming a central Gaussian in the pdf. For a self-similar free shear layer with two infinite supplies of pure fluid, the pdf would reach a steady state (Konrad 1976, Broadwell & Breidenthal 1982). However, if only one fluid supply is infinite, such a finite jet injected into an infinite reservoir, then eventually there is only one delta function in the pdf. If both fluid supplies are finite, then the two delta functions both disappear, and the pdf consists of a central Gaussian, the width of which is the rms concentration fluctuation. As the turbulence further mixes the fluid, the Gaussian progressively narrows and the fluctuations decline.

Here the simplest two assumptions are that both the flow and the mixing are self-similar (Breidenthal *et al.* 1990). The former requires that the vortex rotation period is proportional to its age, as we have seen above. The latter implies that the concentration fluctuations decline by a factor of  $e$  at each rotation. The simple result is that the concentration fluctuations should be proportional to a characteristic time scale  $\tau$  divided by time.

$$\frac{c'}{\bar{c}} = \text{const.} \frac{\tau}{t}. \quad (6)$$

The characteristic time scale is determined by dimensional considerations of the problem. For example, if one fluid is initially in a spherical chamber and a second fluid is momentarily injected into the chamber,  $\tau$  depends on the jet impulse and the chamber diameter. The characteristic time  $\tau$  must also equal the vortex rotation period at the moment  $t = \tau$  when all pure fluid has disappeared and the large-scale vortices have filled the chamber. Measurements of concentration fluctuations are consistent with (6), in spite of the fact that the actual vorticity field appears to decay exponentially instead of as inverse time (Aarnio 1994).

## Density ratio

The coherence of large-scale structure in turbulence was discovered by accident. Brown & Roshko (1974) were attempting to find out about the compressibility effects on entrainment. It was known that supersonic jets exhibited an anomalously low spreading angle. It was not clear if this was due to Mach number or to the density ratio of the supersonic experiments. Since density ratio was easier to control, they elected to measure its effect on spreading angle in incompressible flow by taking shadowgraph pictures. While the most important result of their experiment was the coherent structure revealed by their pictures, they also determined that density ratio has a remarkably weak effect on entrainment rate. The density ratio must vary by a factor of 49 to achieve a factor of two change in spreading angle. This proved that the main influence on jet spreading angle was Mach number.

A simple picture readily accounts for the effect of density ratio on entrainment into a shear layer. Coles (1981) drew the shear layer in the Lagrangian frame of the vortices (see figure 4). Fluid enters a vortex from each stream due to the relative speed of the stream with respect to the vortex. Brown (1974) showed that the relative speed ratio comes from consideration of the stagnation streamlines. Assuming quasi-steady inviscid flow, the total pressure on both streamlines must be constant and equal. Furthermore, the streamlines far from the stagnation point are quasi-parallel, so that their static pressures must be equal. The result is the dynamic pressures of the relative flows far from the stagnation point are equal. So the speed ratio in this frame is just the inverse square root of the density ratio. Dimotakis (1986) neatly summarizes the effects of both density and velocity ratio on both the spreading angle and the entrainment ratio from the two sides of the layer.

### Rotation

Bradshaw (1969) noted that when a fluid rotates, the higher speed fluid tends to want to move to the outside of the turn. This corresponds to a state of lower kinetic energy for the same angular momentum. The difference in kinetic energy between the two states can be dissipated into thermal energy in accord with the second law. On the other hand, if the higher speed fluid is already on the outside of the turn, a rotating flow acts as if it is stratified. This occurs even when the fluid has uniform density. This effective stratification inhibits entrainment.

Cotel (2002) used Bradshaw's analogy to explain the remarkable behavior of aircraft trailing vortices. Even many kilometers behind a large aircraft, the wingtip vortices are compact, laminar cores of only about a meter in diameter, in spite of the large Reynolds number. The radial transport of momentum is strongly inhibited by the effective stratification due to the rotation.

### Stationarity

When a vortex is near a surface, the motion of the vortex with respect to the surface becomes important. The entrainment rate across the surface depends on the amount of stationarity of the vortex. Even a small amount of vortex movement completely changes the physics.

Cotel & Breidenthal (1997 & 1999) first identified this effect at a stratified interface. The entrainment rate across a stratified interface was much different for an impinging vertical jet compared to other turbulent flows, such as from an oscillating grid. The impinging vertical jet entrained fluid across the interface with stationary, lateral vortices, in contrast to the moving vortices from an oscillating grid or horizontal jet.

In order to quantify the stationarity, Cotel defined a new parameter. The persistence parameter  $T$  is essentially the ratio of the rotational to the translational speed of the vortex with respect to the surface (figure 5). When  $T$  is much less than one, the flow is in the nonpersistent limit. When  $T$  is much greater than one, the flow is said to be persistent. For a vortex near a surface, there is no more important parameter than this.



Cotel asserted that the surface may be of any type: a stratified interface, a solid wall, or even an iso-vorticity contour of a neighboring vortex. Thus her theory is applicable to a wide class of flows.

When a piston suddenly begins to push fluid out of a tube at constant velocity, a starting vortex is formed. The subsequent jet never catches up with this vortex ring (Johari *et al.* 1997). If the piston advances sufficiently far, the starting vortex cannot accept all the injected vorticity. Gharib *et al.* (1998) defined a “formation number” to be the ratio of the stroke length to piston diameter. The formation number is essentially identical to the persistence parameter, as noted by Gharib (private communication 1995). There is a transition in vortex behavior at a critical value of the formation number at about four, when the starting vortex ring can no longer accept all the injected vorticity. This transition is important in heart flow.

Another example of persistence is the boundary layer. When the surface is a solid wall, the wall fluxes can be drastically modified by persistence. In order to achieve the persistent limit, strong stationary vortices must be introduced. This is difficult, since a linear vortex near a flat wall is unstable to both short wavelength Widnall (Widnall *et al.* 1974) and long wavelength Crow (1970) instabilities, which would promptly render the vortex nonsteady. Balle (Balle & Breidenthal 2002) suggested that vortices could be stabilized by a wavy wall, substituting for the dividing streamline in a von Karman wake. The wake vortices are known to be at least quasi-stable. Balle found the wall flux measured at the bottom of a trough to be laminar, as predicted by Cotel’s theory. Using flow visualization, Dawson (2005) subsequently confirmed that an otherwise turbulent boundary layer was indeed made laminar by the addition of persistent vortices. However, she found that a small segment of the wavy wall did not achieve laminar flow, due to an adverse pressure gradient in the spanwise direction. It is still an open question if a wall shape can be found that will achieve laminar flow everywhere under persistent vortices. Reducing the heat flux to a laminar value would be useful for turbine blades and hypersonic flow.

Surprisingly, Dawson found that the flow pattern did not correspond to the von Karman wake. Instead, it resembled Kelvin’s cat’s eye flow. As mentioned above, this flow pattern always seems to be associated with laminar fluxes.

These discoveries raise interesting questions about the stabilizing effect of stationary vortices on the flow. It seems reasonable that a stationary vortex would not directly hand off energy into smaller scale eddies, since that presumably requires some kind of nonsteadiness in that vortex. However, the persistent vortex seems to inhibit instabilities even in neighboring vorticity, such as that in the boundary layer below the streamwise vortices. Recent results by Fransson *et al.* (2005) indicate that streamwise vortices can stabilize Tollmein-Schlichting waves.

## Stratification

Based on the persistence parameter, Cotel (Cotel & Breidenthal 1997) proposed a new model for stratified entrainment. It consists of different entrainment regimes, determined by the

Richardson, Reynolds, Schmidt, Prandtl, and persistence parameters. For simplicity, we will only consider the limit of a thin stratified interface.

The Richardson number  $Ri$  (of the largest eddies) is defined as the ratio of the potential to the kinetic energy of the largest eddies at the stratified interface. One can also define the *eddy* Richardson number  $Ri_\lambda$  of a smaller eddy of size  $\lambda$ . For a Kolmogorov spectrum, the eddy Richardson number increases with eddy size.

If  $Ri \ll 1$ , the potential energy is dominated by the kinetic energy and stratification is not important for any eddy. If  $Ri > 1$ , there are at least two possibilities. Depending on the Reynolds number, the smallest eddies at the Kolmogorov microscale  $\lambda_0$  may have an eddy Richardson number  $Ri_{\lambda_0}$  greater than one. If so, then they and therefore all eddies have insufficient kinetic energy to engulf a tongue of fluid across the interface. Consequently, in this limit of strong stratification the interface must be essentially flat. All fluxes are purely diffusive. From dimensional considerations, we can define a corresponding effective entrainment velocity to be the square root of the ratio of the diffusivity divided by some eddy rotation period. The diffusivity corresponds to the flux in question, i.e. mass, momentum, or energy.

There are many choices for the eddy rotation period, ranging from that of the largest to the smallest eddy. Clearly, eddies in the middle cannot be rate limiting, since there is no basis to select one over another. So only the largest or the smallest eddy could be correct. Cotel proposed that in the persistent limit, the correct choice is that of the largest eddy. Remarkably, the fluxes would then be completely independent of any fine-scale turbulence.

While this prediction may not yet have been tested in stratified flow, it does seem to work in the corresponding wall flow discussed above. The heat flux is laminar because the persistent vortices make the flow laminar.

In the non-persistent limit, the fluxes would be controlled by the smallest-scale eddies, corresponding to ordinary turbulent flow. This is in accord with many observations at stratified interfaces and the boundary layer.

If the smallest scale vortices have an eddy Richardson number less than unity, then the interface is not flat. The eddy whose Richardson number is equal to about unity can engulf fluid across the interface. It determines the entrainment rate.

Dramatic evidence of the importance of persistence on stratified entrainment was measured by Cotel *et al.* (1997). Following a suggestion by L. Redekopp (private communication 1995), they tilted an impinging jet and precessed it. The entrainment rate was reduced by orders of magnitude compared to that of the vertical jet. The effect is not only large, but counter-intuitive.

## Conclusions

The entrainment rate of a turbulent flow can always be expressed as the ratio of a length to a time scale corresponding to the entraining eddy. This is a generalization of the entrainment hypothesis of Morton, Taylor, & Turner that accounts for a variety of effects, such as acceleration, compressibility, confinement, stratification, and stationarity.

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### References

Aarnio, M.J. 1994 Mixing by turbulent streamwise vortices confined in a duct, Ph.D. thesis, University of Washington.

Balle, G.J. & Breidenthal, R.E. 2002 Stationary vortices and persistent turbulence in Karman grooves, *J. Turbulence* **3** 33-51.

Bergantz, G.W. & Breidenthal, R.E. 2001 Non-stationary entrainment and tunneling eruptions: A dynamic template for eruption processes and magma mixing, *Geophys. Res. Letters* **28** 3075-3078.

Bhat, G.S. & Narasimha, R. 1996 A volumetrically heated jet: Large-eddy structure and entrainment characteristics, *J. Fluid Mech.* **325** 303-330.

Bogdanoff, D. 1983 Compressibility effects in turbulent shear layers, *AIAA J.* **21** 926-927.

Bradshaw, P. 1969 The analogy between streamline curvature and buoyancy in turbulent shear flow, *J. Fluid Mech.* **36** 177-191.

Breidenthal, R.E., Buonadonna, V.R. & Weisbach, M.F. 1990 Mixing of jets in confined volumes, *J. Fluid Mech.* **219** 531-544.

Breidenthal, R.E. 1992 Sonic eddy - A model for compressible turbulence, *AIAA J.* **30**(1) 101-104.

Breidenthal, R.E. 1999 Turbulent stratified entrainment and a new parameter for surface fluxes, *Recent Research Developments in Geophysical Research*, S.G. Pandalai Ed., Research Signpost, Trivandrum, India, August 1999.

Breidenthal, R.E. 2003 The vortex as a clock, *Advances in Fluid Mechanics*, M. Alam, R. Govindarajan, O.N. Ramesh, & K.R. Sreenivas Ed., Jawaharlal Nehru Centre for Advanced Scientific Research, Bangalore, India.

Broadwell, J.E. & Breidenthal, R.E. 1982 A simple model of mixing and chemical reaction in a turbulent shear layer, *J. Fluid Mech.* **125** 397-410.

- Brown, G.L. 1974 The entrainment and large structure in turbulent mixing layers," *5th Australian Conference on Hydraulics and Fluid Mechanics*, 352–359.
- Brown, G.L. & Roshko, A. 1974 On density effects and large scale structure in turbulent mixing layers, *J. Fluid Mech.* **64** 775-816.
- Coles, D. 1981 Prospects for useful research on coherent structure in turbulent shear flow, *Proc. Indian Acad. Sci. (Engng. Sci.)* **4** 111.
- Corrsin, S. 1974 Limitations of gradient transport models in random walks and in turbulence, *Adv. Geophys.* **18** A, 25.
- Corrsin, S. & Kistler, A.L. 1955 Free-stream boundaries of turbulent flows, *NACA TR 1244*, Washington D.C.
- Cotel, A.J. 2002 Turbulence inside a Vortex – Take Two, *Physics of Fluids*, **14** (8) 2933.
- Cotel, A.J. & Breidenthal, R.E. 1997 Persistence effects in stratified entrainment, *Applied Scientific Research* **57** 349-366.
- Cotel, A.J. & Breidenthal, R.E. 1999 Vortex persistence - A recent model for stratified entrainment and its application to geophysical flows, *Geophysical Flows*, Kluwer.
- Cotel, A.J., Gjestvang, J.A., Ramkhelawan, N.N. & Breidenthal, R.E. 1997 Laboratory experiments of a jet impinging on a stratified interface *Exp. Fluids* **23** 155-160.
- Crow, S.C. 1970 Stability theory for a pair of trailing vortices, *AIAA J.* **8** 2172-2179.
- Dimotakis, P.E. 1986 Two-dimensional shear-layer entrainment, *AIAA J.* **21** 1791. I shear-layer entrainment, *AIAA J.* **21** 1791.
- Dimotakis, P.E. 2005 Turbulent mixing, *Annual Reviews of Fluid Mechanics* **37** 329-356.
- Dawson, O.R. 2005 The effect of persistent vortices on boundary layer behavior in flow along a wavy wall, M.S. thesis, University of Washington.
- Finson, M.L. 1973 Hypersonic wake aerodynamics at high Reynolds numbers, *AIAA J.* **11**(8), 1137-1145.
- Fransson, J.H.M., Brandt, L. Talamelli, A. & Cossu, C. 2005 Experimental study of the stabilization of Tollmein-Schlichting waves by finite amplitude streaks, *Phys. of Fluids* **17**(054110).
- Gharib, M., Rambod, E. & Shariff, K. 1998 A universal time scale for vortex ring formation, *J. Fluid Mech.*, **360** 121-140.

Johari, H., Zhang, Q., Rose, M. & Bourque, S., 1997 Impulsively started turbulent jets, *AIAA J.* **35**(4) 657-662.

Kato, S.M., Groenewegen, B.C. & Breidenthal, R.E. 1987 On turbulent mixing in nonsteady jets, *AIAA J.*, **25**(1) 165-168.

Kelvin, Lord (Thomson, William T.) 1880 On a disturbing infinity in Lord Rayleigh's solution for waves in a plane vortex stratum, *Nature* **23**, 45-46.

Konrad, J.H. 1976 An experimental investigation of mixing in two-dimensional turbulent shear flows with applications to diffusion-controlled chemical reactions, Ph.D. thesis, California Institute of Technology; and *Project SQUID Tech. Rep.* CIT-8-PU.

Morton, B.R., Taylor, G.I. & Turner, J.S. 1956 Turbulent gravitational convection from maintained and instantaneous sources, *Proc. Roy. Soc. A* **234** 1-23.

Oster, D. & Wignanski, I. 1982 The forced mixing layer between parallel streams, *J. Fluid Mech.* **123** 91-130.

Papamoschou, D. & Roshko, A. 1989 The compressible turbulent shear layer: An experimental study, *J. Fluid Mech.* **197** 453.

Roberts, F.A. 1985 Effects of a periodic disturbance on structure and mixing in turbulent shear layers and wakes, Ph.D. thesis, California institute of Technology.

Roshko, A. 1976 Structure of turbulent shear flows: A new look, *AIAA J.* **14**(10) 1349-1353.

Widnall, S.E., Bliss, D.B. & Tsai, C-Y 1974 The instability of short waves on a vortex ring, *J. Fluid Mech.* **66** 35-47.

Zhang, Q. & Johari, H. 1996 Effects of acceleration on turbulent jets, *Physics of Fluids*, **8**(8) 2185-2195.

## Figure captions

- Figure 1. Entrainment velocity  $v_e$  is proportional to the thermal rise speed  $W$  according to the entrainment hypothesis (Morton *et al.*)
- Figure 2. Temporal evolution of the vortex rotation period for self-similar flow
- Figure 3. Probability density function of the concentration field of a passive scalar is composed of contributions from the pure fluid, Taylor layers, and the vortex cores (Broadwell)
- Figure 4. Sketch of the flow in the shear layer for an observer moving with the vortices (Brown, Coles, and Dimotakis)
- Figure 5. The intrinsic velocity ratio of a vortex near a surface – vortex persistence  $T = U_2/U_1$  (Cotel)
- Figure 6. Cat's eye flow (Kelvin)
- Figure 7. Stratified entrainment diagram in the persistent limit (Cotel)
- Figure 8. Stratified entrainment diagram in the nonpersistent limit (Cotel)

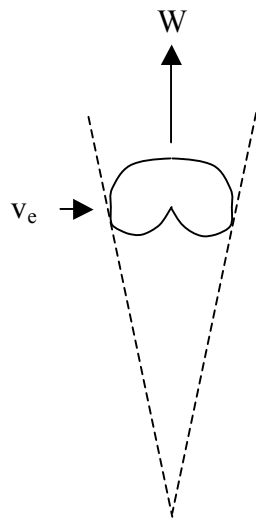


Figure 1

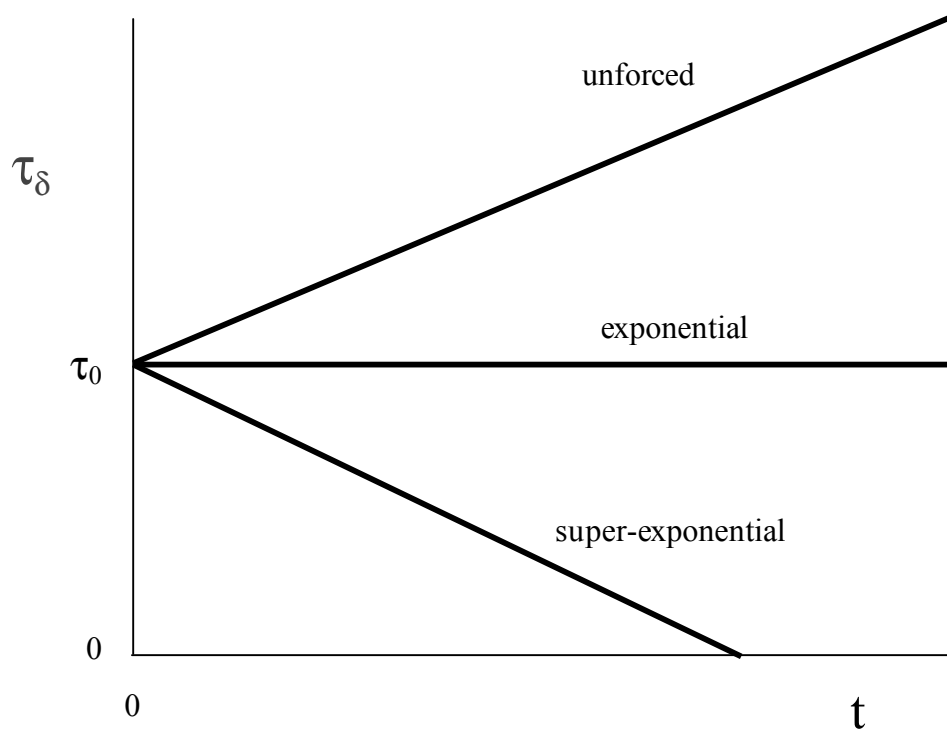


Figure 2

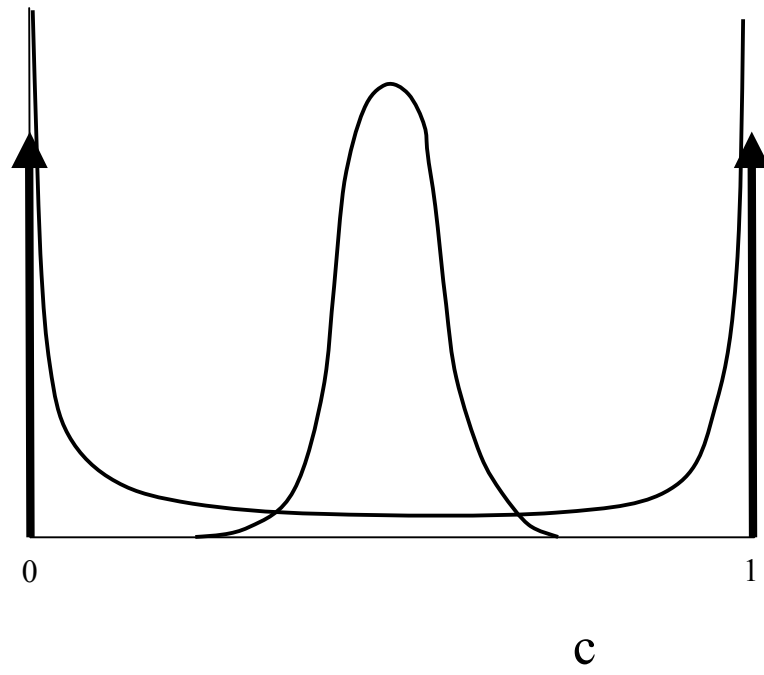


Figure 3

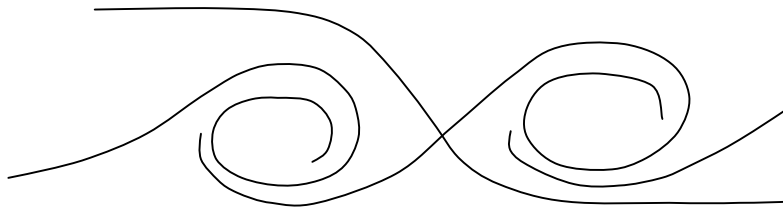


Figure 4



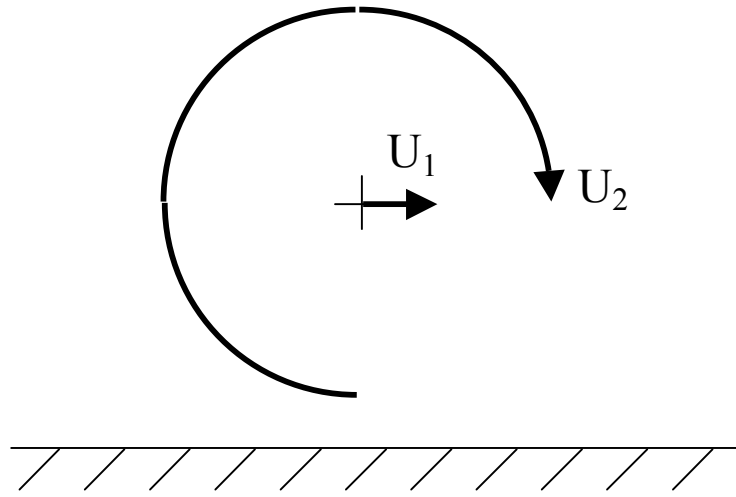


Figure 5

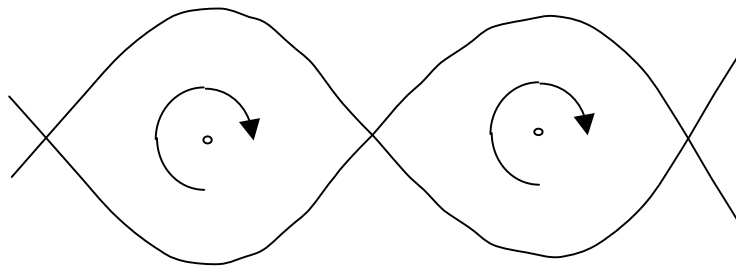


Figure 6

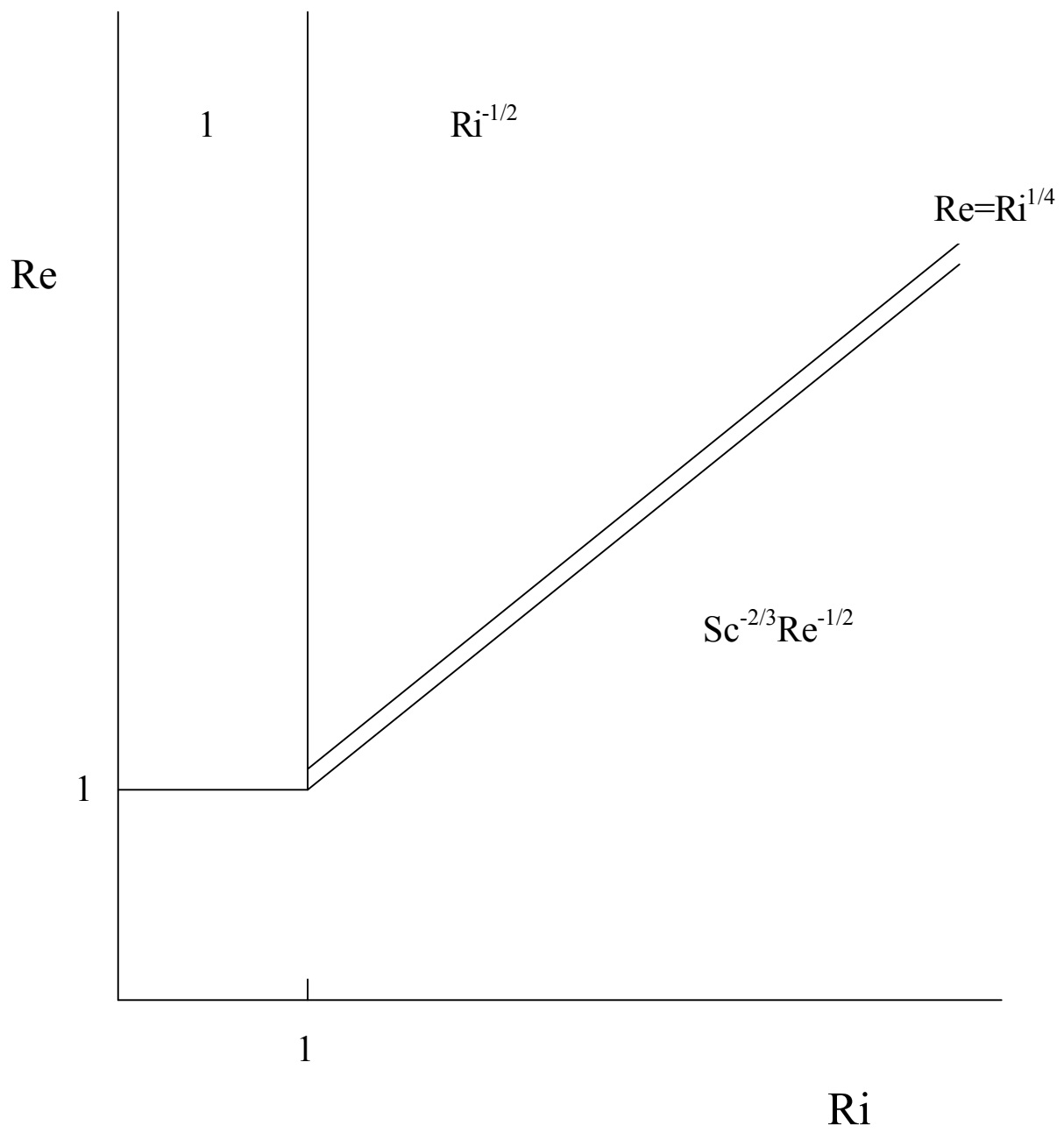


Figure 7

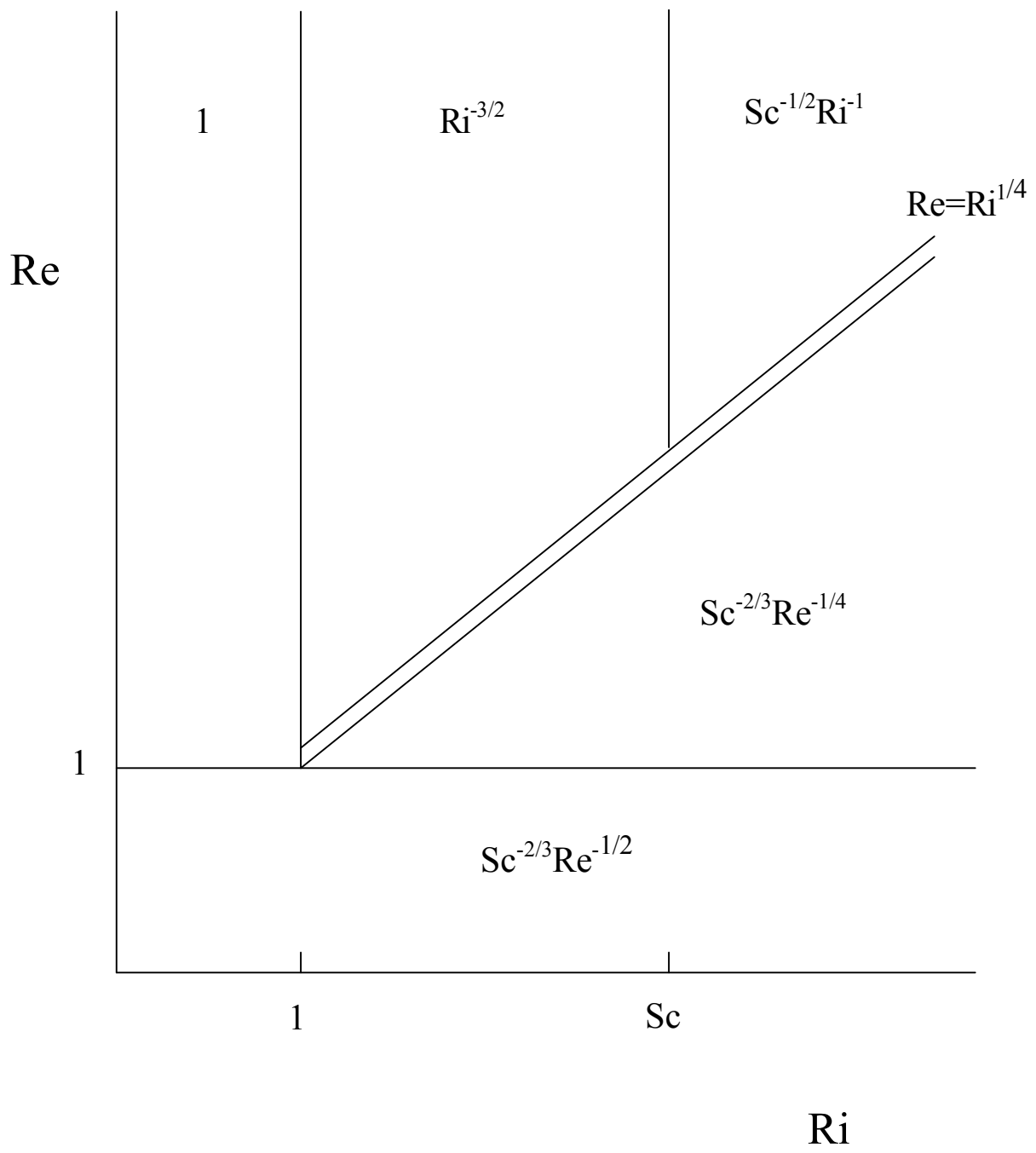


Figure 8